Air Heater Thermal and Flow Profile Analysis

Contents

Introduction .................................................................................................................................................. 2
Flow Distribution Grid Design ................................................................................................................ 3
Thermal Analysis ...................................................................................................................................... 3
   Method .................................................................................................................................................. 3
   Results ................................................................................................................................................ 8
Introduction

This report outlines the thermal analysis and airflow distribution model as part of an air heater package produced by Zeeco. The first section includes design details of the flow distribution grid applied in the inlet of the heater, located below the duct burners. The grid design was based on a CFD analysis.

The second section includes an analysis performed by Zeeco to determine the final temperature distribution at the exit of the heater. This was accomplished using a program to perform a thermal balance based on flow and temperature on each element. These elements are created by cutting the heater into a number of cross sections at different elevations and overlaying a grid on each section.
Flow Distribution Grid Design

The flow distribution grid is a device inserted into the inlet of the heater to create a uniform velocity distribution for combustion. In this particular case, air enters the heater after a 90° bend from a horizontal to vertical direction. This change in direction causes a low static pressure region on the inside of the bend, resulting in a variance of velocity and turbulent flow.

The flow distribution grid utilizes five “zones” with different percentages of open areas that restrict flow in high pressure areas and create a uniform velocity distribution across the outlet of the grid. The design goals are as follows:

- Redistribute the air to ensure a velocity distribution of ±5% at the exit of the grid
- Ensure the total pressure drop across the vessel is less than 1 PSI

Thermal Analysis

The objective of the thermal analysis was to determine the final temperature distribution at a cross section located at the outlet of the air heater and controlled heat release (drilling pattern). This ensures that the temperature variance was minimized to meet the specification. Calculations were performed by dividing each cross section into 6”x6” square elements in a grid pattern. This determined the temperatures performed for each element at cross sections every 6” in elevation above the burners, starting at the level of fuel injection. The provided drilling was used to find the heat released at each cell. Calculations were performed until a satisfactory temperature variance was indicated or the top of the heater was reached. The temperature calculations process included taking three iterations at each cross section before the final temperatures were recorded.

Method

The following list outlines each step of the calculation process to find the final temperature distribution in order:

1. **Initial temperature**

   The initial temperature listed for each cell is the final temperature from the cross section 6” and below. This temperature is used to calculate the specific heat, \( C_p \). The amount of heat added to the cell will generate a temperature rise. This will be added to this initial temperature to find the 0\(^{th} \) iteration final temperature.
2. \textit{C}_p \textit{calculation}

The specific heat is used to calculate the 0\textsuperscript{th} iteration final temperature. \textit{C}_p was tabulated based on the initial air temperature for each cell.

3. \textit{Heat Released}

The heat released varies for each cell and elevation. The tip drilling pattern defines the total heat release over the entire flame height for each cell. Once this was determined, the heat released for a particular cell at each elevation was calculated using the following equation:

\[
HR_z = HR_{\text{total}} \frac{Z}{Z_{\text{max}}} - HR_{z-6}
\]

HR= heat released  
Z= height of cross section  
\(Z_{\text{max}}\) = total flame length for the cell in question

Note: the heat released is the heat \textit{added} between the previous elevation to the current elevation.

4. 0\textsuperscript{th} iteration final temperature

The temperature at this point for each cell is the initial temperature including the rise due to the heat added by the flame. The formula used is the simplified steady-flow thermal equation as shown below:

\[
q = HR_x = m_{\text{dot}}c_p(T_{\text{out}} - T_{\text{in}})
\]

This formula assumes all heat entering a cell results in a change of internal energy in the form of higher temperature. If there is no heat added to a cell, the initial temperature is simply carried over.
Figure 1: Initial grid with line of symmetry

Note the grid numbering system. Due to the symmetry of the velocity contour plot, calculations were only performed for the east portion of the heater.

5. First Iteration final temperature

To model the transfer of heat from hot to cooler grids, cells were combined and averaged, creating a grid of larger squares (12”x12” maximum in Iteration 1). To account for cells around the perimeter with smaller areas and cells with different mass flows, the average was weighted accordingly.

This method assumes the grids have approximately the same compositions and specific heats so that only the cell areas, flow rates, and temperatures factor into the rate of heat transfer to adjacent cells.

To find the mass flow rates through each cell, the velocity contour plot was mapped and entered into each cell. This contour plot was taken at a cross section 14” below the runners in order to avoid localized airflow effects. The velocities for each cell were then averaged and weighted by area to find the overall average velocity. The percentage variance from the average velocity was then obtained from each cell to create a mass flow correction factor. This factor is multiplied by the average mass flow per cell to find the actual mass flow through each cell.

\[
\frac{2,491,200 \text{ LBS}}{688 \text{ Total cells}}
\]

The mass flow of smaller cells was also corrected.
The formula to obtain the final temperature for Cell 2, Iteration 1 (Cell 1.2) is as follows:

\[
T_{0,1.2} = \frac{m_{dot,2}A_2 T_2 + m_{dot,3}A_3 T_3 + m_{dot,32}A_{32} T_{32} + m_{dot,33}A_{33} T_{33}}{m_{dot,2} A_2 + m_{dot,3} A_3 + m_{dot,32} A_{32} + m_{dot,33} A_{33}}
\]

In this case, since the areas are equal they cancel out.

6. **Second Iteration of final temperature**

Calculation of the final temperature of this iteration follows the same procedure as the previous iteration but with the grid now being spaced 18” apart.
7. Third Iteration of final temperature

In this case, the grid is 24”x24”, so the grid from the first iteration was used as the basis to average in order for simplifying calculations. Iteration 3 was the final iteration, yet adjacent cells still were not able to transfer heat to each other. To solve this, Iteration 3b was created, which shifts the grid south 6” and east 6”. The method of calculation was switched at every cross section, i.e.: cross section 1 used 3a, cross section 2 used 3b, cross section 3 used 3a, etc.

The final temperature from Iteration 3 or 3b was the final calculation for a cross section.
Results

After the thermal analysis and airflow distribution model results have been reviewed, the heater temperature distribution is reported is sufficiently uniform (less than 1% variance from average) when the heater begins to neck down at the outlet. This result is due to the increased air mixing and time to reach equilibrium.

The following charts take the temperatures along several chords parallel to the North-South line of symmetry at several elevations. The elevations chosen are all at 12” apart so that all charts utilize the same formula for the 3rd Iteration. Z=264” is the final elevation shown, just below the point where the heater begins to neck down at the outlet.
Figure 6: Average temperature (area weighted) at each elevation
Figure 7: Final Temperature Distribution at Z=270"

Largest 1353.199 F
Smallest 1335.492 F
Average 1346.098 F

Variance 0.787854 % (variance from average)
10.60529 F (variance from average)

Figure 8: Temperature distribution at Z=270"